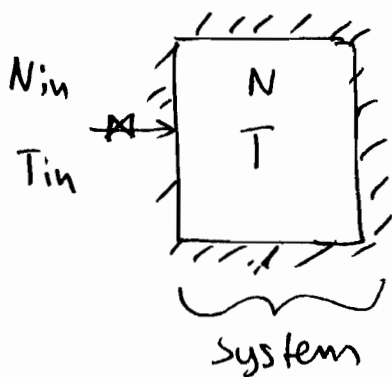


Problem Solving Strategy

1. Draw a schematic diagram, label key quantities
2. Define the thermodynamic system(s) of interest
3. State assumptions
4. Write down first Law balance
5. Apply equation-of-state (ideal gas) equation or obtain properties from NIST webBook
6. Solve using symbols as far as possible

Example 2.5 - filling of gas cylinder



Insulated system, N_i, T_i

Pump in gas at T_{in} until

N_f moles in tank

$$T_f = ?$$

* assume ideal gases, contents of cylinder at equilibrium as it is being filled

* $Q=0$ (insulated) $W=0$

$$dU = \underline{H}_{in} dN_{in} \quad \left[\begin{array}{l} U = N\underline{u}, \text{ both } N \text{ and} \\ \underline{u} \text{ are changing!} \end{array} \right]$$

$$d(N\underline{u}) = N d\underline{u} + \underline{u} dN = \underline{H}_{in} dN$$

↑
 $dN_{in} = dN$

For balances involving ideal gases, common (easiest) reference state to use is $\underline{u}=0$ at $T=0$

$$\Rightarrow \underline{H}=0 \text{ @ } T=0. \text{ Then } \underline{u} = C_v T \text{ (const. } C_v)$$

$$\underline{H} = C_p T \text{ (const. } C_p)$$

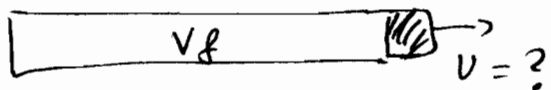
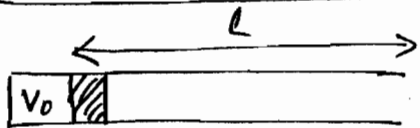
Apply here: $N C_v dT + C_v T dN = C_p T_{in} dN \Rightarrow$

$$\Rightarrow N C_v dT = (C_p T_{in} - C_v T) dN \Rightarrow$$

$$\int_{T_i}^{T_f} \frac{dT}{\delta T_{in} - T} = \int_{N_i}^{N_f} \frac{dN}{N} \Rightarrow -\ln \frac{\delta T_{in} - T_f}{\delta T_{in} - T_i} = \ln \frac{N_f}{N_i} \Rightarrow$$

$$\Rightarrow \frac{\delta T_{in} - T_f}{\delta T_{in} - T_i} = \frac{N_i}{N_f} \Rightarrow T_f = \delta T_{in} - \frac{N_i}{N_f} (\delta T_{in} - T_i)$$

Example 2.8 - Air rifle



$$V_0 = 10 \text{ cm}^3$$

$$P_0 = 3 \text{ bar (absolute)}$$

$$T_0 = 25^\circ \text{C} \Rightarrow T_0 = 298 \text{ K}$$

$$m = 2 \text{ g} \quad A = 0.3 \text{ cm}^2$$

$$l = 50 \text{ cm}$$

$$C_v^{\text{air}} = 20.8 \text{ J/(mol K)} = \frac{5R}{2}$$

$$C_{v, \text{proj}} = 0.126 \text{ J/(g K)}$$

$$P_{\text{atm}} = 1 \text{ bar}$$

$$V_f = V_0 + A \cdot l = 25 \text{ cm}^3$$

Rapid expansion \rightarrow adiabatic, Eq. (2.30)

$$\left(\frac{T_f}{T_0}\right) = \left(\frac{V_f}{V_0}\right)^{-\frac{R}{C_v}} \Rightarrow T_f = 298 \left(\frac{25}{10}\right)^{-\frac{2}{5}} \text{ K} = 207 \text{ K}$$

Work: $\Delta u = \cancel{Q} + w \Rightarrow \Delta u = w \Rightarrow$

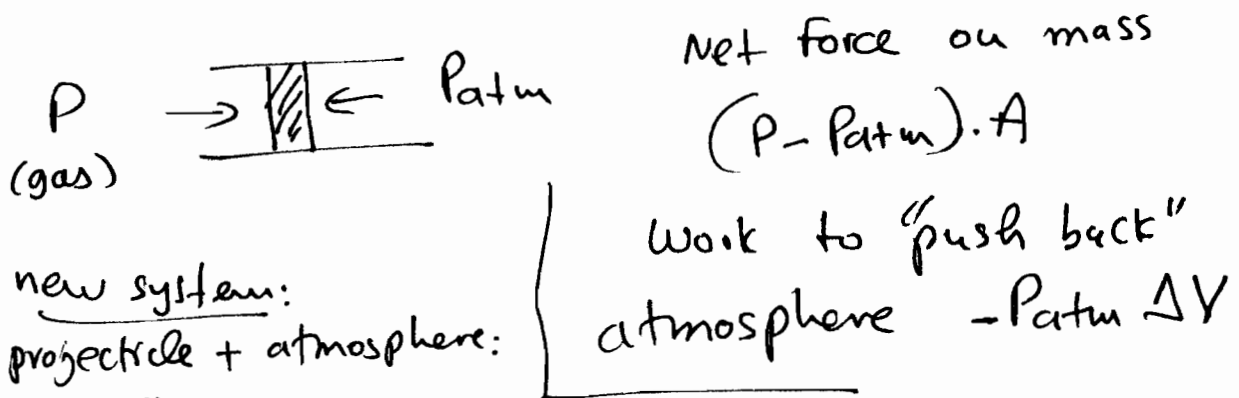
$$w = N c_v \Delta T = \frac{P_0 V_0}{R T_0} c_v \Delta T =$$

$$= \frac{3 \cdot 10^5 \text{ Pa} \cdot 10 \cdot 10^{-6} \text{ m}^3 \cdot 5/2}{298 \text{ K}} \cdot (207 - 298) \text{ K} =$$

$$= -2.30 \text{ (Pa} \cdot \text{m}^3) = -2.30 \text{ J} \quad (\text{produced by gas})$$

[PV has units of work]

But... not all of this work goes into kinetic energy of the projectile



(new $w = -$ old w)
 (new $\Delta V = -$ old ΔV)

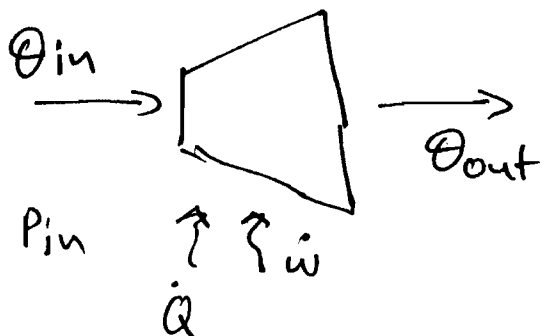
$$w = \frac{1}{2} m v^2 - P_{atm} \Delta V \Rightarrow$$

$$v = \sqrt{\frac{2(w + P_{atm} \Delta V)}{m}} = \sqrt{\frac{2 \cdot (2.30 \text{ J} - 10^5 \text{ Pa} \cdot 15 \cdot 10^{-6} \text{ m}^3)}{0.002 \text{ kg}}}$$

$$\Rightarrow v = 28.3 \text{ m/s}$$

For projectile hitting a wall, assume all kinetic energy goes to heat it, $\Delta u = m c_v \Delta T = \frac{1}{2} m v^2 \Rightarrow$

$$\Delta T = 3.2 \text{ K}$$

Example 2.9 Turbine Power

$$\dot{Q} = -3 \text{ kW}$$

$$T_{in} = 200^\circ\text{C}$$

$$P_{in} = 3 \text{ bar}$$

$$\dot{N} = 0.12 \text{ kg/s}$$

$$T_{out} = 140^\circ\text{C}, \text{ saturated vapor}$$

Steady-state:

$$\frac{dU}{dt} = 0 = \dot{Q} + \dot{W} + \dot{N}(\underline{H}_{in} - \underline{H}_{out}) \Rightarrow$$

$$\dot{W} = -\dot{Q} + \dot{N}(\underline{H}_{out} - \underline{H}_{in})$$

$$\left. \begin{array}{l} \underline{H}_{out} \text{ (satd., } 140^\circ\text{C)} = 2733.4 \text{ J/kg} \\ \underline{H}_{in} \text{ (3 bar, } 200^\circ\text{C)} = 2865.9 \text{ J/kg} \end{array} \right\} \text{ from WebBook}$$

$$\dot{W} = +3 \text{ kW} + 0.12 \frac{\text{kg}}{\text{s}} \cdot (2733.4 - 2865.9) \frac{\text{J}}{\text{kg}} \Rightarrow$$

$$\boxed{\dot{W} = -12.9 \text{ kW}}$$